HPC Computer Aided Engineering @ CINECA

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Outline

• What is CFD
• CFD main concepts
• CFD main definitions
• How should I use it
Computational Fluid Dynamics: CFD

- Fluid dynamics: physics of fluids
- Computational: numerical engine involved in solving the equation describing the motion of the fluid by means of iterative methods

There is a strong interplay between math concepts, physics knowledge and technological tools and environments used to implement a CFD model.

Demanding task:
- No general rules for specific CFD model setup
- Need of a-priori knowledge of the fluid behavior
- Need of a-priori knowledge of the physics involved by the problem
- If possible experimental (or theoretical) data to validate CFD results
CFD workflow

**PRE-PROCESSING**

- PHYSICAL MODEL
- COMPUTATIONAL MODEL

**COMPUTATION**

- SOLVING: HPC ENVIRONMENT

**POST PROCESSING**

- VISUALIZATION: RESULTS
CFD: general approach

A very rough definition to explain how CFD works can be as follow:

“CFD applies the principles of conservation of mass and momentum according to the geometry and the mechanical properties of the fluid involved by the problem to be able to get information on instantaneous velocity and pressure distributions.”

Three pylons:
1. Geometry description
2. Fluid properties
3. Fluid conditions
CFD: general idea

Analytical (Exact) solution (integrals) is not available for the specific geometry (CAD) but thanks to iterative (Automated) methods a numerical (Approximated) solution of the problem can be found.

Continuum $\rightarrow$ Discrete
Math $\rightarrow$ Numeric
Numeric $\leftrightarrow$ Technology (HW, SW)

When moving from math to numeric we need to understand that there is a counterpart version of each concept and that once we move to numeric also technical issues comes in to play.
CFD: general approach

When moving **from math to numeric** we need to understand that there is a counterpart version of each concept and that once we move to numeric also technical issues comes in to play.

Math

- Convergence
- Consistency
- Stability

||

Numeric

- Boundedness
- Conservativeness
- Transportiveness
Problem Discretization

Theory → Practice

Theory:

IF \#cells (used to discretize the continuum) → \infty
THEN the numerical solution → ‘exact’
and this will be independent from the numerical scheme adopted.

Practice:

IF \#cells if finite
THEN the numerical solution → ‘OK’
and this will be dependent from the properties of the numerical scheme adopted.

This part of the workflow is the so-called pre-processing and/or meshing process
CAD-Mesh-Cell

Computational domain  Mesh/grid/discretized domain  Cell
CAD-Mesh-Cell

• Domain is discretized into a finite set of control volumes or cells. The discretized domain is called the “grid” or the “mesh.”
• General conservation equations for mass and momentum are discretized into algebraic equations and solved for each and all cells in the discretization.
• (physic --> math --> numeric --> sw)
The finite-volume method (FVM) is a method for representing and evaluating partial differential equations in the form of algebraic equations [LeVeque, 2002; Toro, 1999]. Similar to the finite difference method or finite element method, values are calculated at discrete places on a meshed geometry. "Finite volume" refers to the small volume surrounding each node point on a mesh. In the finite volume method, volume integrals in a partial differential equation that contain a divergence term are converted to surface integrals, using the divergence theorem. These terms are then evaluated as fluxes at the surfaces of each finite volume. Because the flux entering a given volume is identical to that leaving the adjacent volume, these methods are conservative. Another advantage of the finite volume method is that it is easily formulated to allow for unstructured meshes. The method is used in many computational fluid dynamics packages.

[Wikipedia]
Finite volume method

• In order to build a CFD model of a physical problem only the portion of your problem where the fluid is present need to be discretized (meshed).
• CFD models usually are used to study local phenomena related to a 3D problem so usually only a a certain amount of your physical domain need to be included in the CFD model (‘nearby’)
• All other parts (solid or ‘far away’) are not of interest and are not included in the meshing process.
• Example Wind Tunnel Application: the solid object surrounded by the flowing air is not meshed except for his surface.
• All other surfaces are called numerical boundaries (or just boundaries) and can be described efficiently within the CFD model by understanding their main characteristics.
In order to build a CFD model correctly you must select, accordingly to your available experimental data or your a-priori knowledge, a coherent set of boundaries (so that you solve a problem that results in a unique solution)

Activities involved are:

1. identify the position of the boundaries in your problem
2. recognize what kind of boundary you are dealing with (inlets, outlets, walls, symmetry, cyclic…)
3. retrieve the information available from your physical problem for that boundary
Finite volume method

Boundary conditions are a necessary part of the mathematical model. In fact Navier-Stokes equations and continuity equation in order to be solved need:

- initial conditions (starting point for the iterative process)
- boundary conditions (define the flow regimen problem)

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Neumann and Dirichlet boundary conditions

• Dirichlet boundary condition:
  Value of velocity at a boundary
  \[ u(x) = \text{constant} \]

• Neumann boundary condition:
  gradient normal to the boundary of a velocity at the boundary,
  \[ \partial_n u(x) = \text{constant} \]
Pressure BC convention

- Pressure boundary conditions require static gauge pressure inputs:

\[ P_{\text{absolute}} = P_{\text{static}} + P_{\text{operating}} \]

- An operating pressure input is necessary to define the pressure (0 will work well).
Pressure outlet boundary

- Pressure outlet BC can be used in presence of velocity BC at the inlet
- Usually in a zero-stress condition is applied over multiple exits (if not known specifically from a-priory knowledge of the problem)
- The geometry is driving the pressure distribution and the flow repartition
- The static pressure is assumed to be constant over the outlet
Velocity inlets

- A flat profile is selected by default (if not known specifically from a-priory knowledge of the problem).
- Is very often used as BC to set a known flow-rate (working condition)
- Thanks to experimental data is possible to obtain spatial and temporal distribution.
Wall boundaries

• Used to bound the limit between fluid and solid regions in our problem
• Settings at the wall:
  – Tangential fluid velocity equal to wall velocity (usually zero))
  – Normal velocity component is set to be zero.
Symmetry

• When your problem presents a symmetry plane in the geometry and in the flow field, this kind of boundary can be used to reduce computational cost of the CFD model.

• General characteristics at the symmetry plane:
  – normal velocity equal to zero
  – normal gradients of all variables equal to zero
Material properties

• The physical property of the fluid must be given.
• For Newtonian Incompressible fluid we have to provide only density and viscosity.
• Newtonian?
• Incompressible?
Density

- Density is a fluid property and is defined as:
  \[ \rho = \frac{\text{Mass}}{\text{Volume}} \]

- Usually we consider for water the density value at \( T=300 \text{ K} \) and \( P=105 \text{ Pa} \):
  \[ \rho = 1060 \text{ [Kg/m}^3\text{]} \]
Viscosity

• For viscous flow if the relationship between viscous forces (tangential component) and velocity gradient is linear then the fluid is called Newtonian and the slope of the line is a measure of a fluid property called viscosity (dynamic):
  \[ \text{Sy} = \mu \frac{dv}{dx} \]

• Also a cinematic viscosity can be defined by:
  \[ \upsilon = \frac{\mu}{\rho} \]
Reynolds number

The Reynolds number Re is defined as:

\[ \text{Re} = \frac{\rho \ U \ L}{\mu} \]

Here:

- \( L \) is a characteristic length (say \( D \) in tubes)
- \( U \) is the mean velocity over the section \((Q/\text{Area})\)
- density and viscosity are: \( \rho, \mu \)

If \( \text{Re} \gg 1 \) the flow is dominated by inertia.
If \( \text{Re} \ll 1 \) the flow is dominated by viscous effects.

\[ \text{Re} = \frac{\text{convective inertia force}}{\text{viscous friction force}} \]
Flow classifications

Laminar vs. turbulent flow.
- Laminar flow: fluid particles move in smooth, layered fashion (no substantial mixing of fluid occurs).
- Turbulent flow: fluid particles move in a chaotic, “tangled” fashion (significant mixing of fluid occurs).

Steady vs. unsteady flow.
- Steady flow: flow properties at any given point in space are constant in time, e.g. \( p = p(x,y,z) \).
- Unsteady flow: flow properties at any given point in space change with time, e.g. \( p = p(x,y,z,t) \).
Incompressible vs. compressible flow

- Incompressible flow: volume of a given fluid particle does not change.
  - Implies that density is constant everywhere.
  - Essentially valid for all liquid flows.
- Compressible flow: volume of a given fluid particle can change with position.
  - Implies that density will vary throughout the flow field.
Single phase vs. multiphase flow &
homogeneous vs. heterogeneous flow

- **Single phase flow**: fluid flows without phase change (either liquid or gas).
- **Multiphase flow**: multiple phases are present in the flow field (e.g. liquid-gas, liquid-solid, gas-solid).
- **Homogeneous flow**: only one fluid material exists in the flow field.
- **Heterogeneous flow**: multiple fluid/solid materials are present in the flow field (multi-species flows).
Working hypothesis in CFD for general fluids

- Continuum hypothesis
- Homogenous
- Incompressible (density $\rho$ is constant)
- Isotropic (same behaviour in all directions)
- Newtonian (viscosity ($\mu$ and $\lambda$) are constant and do not depend on the shear rate)

In physical models it is very hard to guarantee all these hypothesis (repeatability and calibration)
Equations of conservation

Two general conservation equations:
1. Mass (divergence free)
2. Momentum: Newton’s second law: the change of momentum equals the sum of forces on a fluid particle

Control-volume: mesh cell
The rate of change over time (local acceleration) + Transport by convection = Pressure forces + Diffusion/viscous forces + Source terms

1. Non linear
2. Coupled (also in the continuity eqn)
3. Role of pressure (no equation of state for Newtonian incompressible fluids)
4. Second order derivatives
Iterative methods

All the issues abovementioned require numerical methods and techniques to build accurate and stable tools to integrate such equations.
Guessed values and relaxation

- The iterative method to move from one iteration to the other uses guessed values.
- New values are found using the old value and a guessed one according to:

\[ \phi_{P}^{\text{new, used}} = \phi_{P}^{\text{old}} + U(\phi_{P}^{\text{new, predicted}} - \phi_{P}^{\text{old}}) \]

Here U is the relaxation factor:
- U < 1 is underrelaxation.
- U = 1 corresponds to no relaxation. One uses the predicted value of the variable.
- U > 1 is overrelaxation.

Interpolation schemes are related to the way that I use to build my used value
Conservativeness

The conservation of the fluid property $\phi$ must be ensured for each cell and globally by the algorithm.
Boundedness

Iterative methods start from a guessed value and iterate until convergence criterion is satisfied all over the computational domain.

In order to converge math says that:

1. Diagonal dominant matrix (from the sys. of eqn.)
2. Coefficients with the same sign (positive)

Physically this means:

1. If you don’t have source terms the values are bounded by the boundary ones (if the pb is linear).
2. If a property increases its value in one cell then the same property must increase also in all the cells nearby.

Overshoot and undershoot present for certain algorithms is related to this property
Transportiveness

Directionality of the influence of the flow direction must be ‘readable’ by discretization scheme since it affects the balance between convection and diffusion.

So called ‘false-diffusion’ (i.e. numerically induced) is related to this property
Pressure - velocity coupling

- For incompressible N-S eqn there is no explicit equation for $P$.
- $P$ is involved in the momentum equations.
- $V$ must satisfy also continuity equation.

The so-called ‘pressure-velocity’ coupling is an algorithms used to obtain a valid relationship for the pressure starting from the momentum and the continuity equation.

The oldest and most popular algorithm is the SIMPLE (Semi-Implicit Method for Pressure-Linked Equations) by Patankar and Spalding 1972.
What is convergence

Convergence: where do I stop my iterations?

• A flow field solution is considered ‘converged’ when the changes of the properties in the cells from one iteration to another are below a certain fixed value.

• General laws are missing; we have some good rules to understand when we are converged and we can stop to iterate over our solution.
Residuals

- Residual: \( R_p = |a_P \phi_P - \sum_{nb} a_{nb} \phi_{nb} - b| \)
- Usually scaled and normalized

#1
Residual control during computation
Other convergence criteria

#2
Monitor ‘the other‘ quantity on boundaries

#3
Monitor changes on quantities you are interested on
Post-processing & data visualization

Once the CFD model is ‘at convergence’ the data concerning the flow field in the discretized domain are available in order to be processed for quantitative analysis and for visualization.

Scientific visualization over CFD data is one of the most interesting characteristics of this tool because it allows designers and engineers to get a better understand of the physical phenomenon and so to improve the insight on it.

Once that P and U fields at each point in the domain are available it is possible to perform further processing and extract no matter which derived quantity at any point with an accuracy that is not reachable by common experimental measurements techniques.
Source of errors and uncertainty

• Error: deficiency in a CFD model.
  Possible sources of error are:
  • Numerical errors (discretization, round-off, convergence)
  • Coding errors (bugs)
  • User errors

• Uncertainty: deficiency in a CFD model caused by a lack of knowledge.
  Possible source of uncertainty are:
  • Input data inaccuracies (geometry, BC, material properties)
  • Physical model (simplified hypothesis for the fluid behavior)
Verification and Validation (V&V)

Verification: “solving the equation right” (Raoche ‘98).
This process quantify the errors.

Validation: “solving the right equations” (Roache ‘98).
This process quantify the uncertainty.
How should I use it

CFD is a engineering **design tool** and like any tool can be used with success only were is **suited** to solve a certain problem and by people with sufficient **knowledge** on how to use the tool.

![Right tools](image)

- **Your problem**

- **Wrong tool**